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A complete mathematical modelling of robots actuated by epicyclic-gear

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Abstract

The industrial robots often use planetary gear system to have high joint torques; therefore, the influence of the rotary inertia of the number of the equally spaced planet-gears on the dynamical behavior of the robot is very important. The main objective of this paper is to develop the dynamic modeling of robot actuated by *(n)* equally spaced planet-gears in the case where the planet-carrier is fixed, no closed solution has been reported for this dynamic modeling, and to compare between the dynamic behavior of robot actuated by *(n+1)* and *(n)* equally spaced planet-gears for a same trajectory planning. The authors derive the explicit dynamic model for an elbow down of 2-R manipulator holding an external mass. Finally, the obtained simulation results of the dynamic modeling are verified by modeling the same robot and using an advanced simulation via *SolidWorks.*

Keywords: Dynamic Modelling; (n) Equally Spaced Planet-Gears; Kinetic Energy; Joint Torque

1. Introduction

Industrial Robots are active systems that require a source of energy to power all their functions. The energy needed for operation must be distributed to the various functions and opportunely modulated, by power converters, which are themselves managed by a suitable low level controller. The power converters provide energy to actuators that transduce the electrical energy supplied by the source into the mechanical energy needed to perform the various tasks [1, 2, 3, 4]. The actuators provide power to act on the mechanical structure against gravity, inertia, and other external forces to modify the pose of the robot's hand [5].

To choose the components of an actuating system, it is worth starting from the requirements imposed on the mechanical power by the force and velocity that describe the joint motion [6].

The execution of joint motions of a manipulator demands low speeds with high torques [6]. In general, such requirements do not allow an effective use of the mechanical features of electrical motors, which typically provide high speeds with low torques in optimal operating conditions. It is then necessary to interpose a transmission (gear) to optimize the transfer of mechanical power from the motor to the joint. During this transfer, the power is dissipated as a result of friction.

The use of gearing is a well-established means for accomplishing such an objective because it is possible to use smaller actuators to deliver larger torques [7].

However, one of the biggest disadvantages of the addition of gearing is the presence of backlash that reduces the effective bandwidth of the controller since high frequency content in the control produces noisy and destructive operation [8].

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Most industrial robot manipulators are driven by motors through gears with high reductions ratios (from tens to a few hundreds). The use of gears permits an optimization of manipulator static and dynamic performance since the motors can be located on the link preceding the actuated joints along the kinematic chain. Further, typical robot applications require motions with large torques and relatively small velocities, and thus the use of gears allows joint actuation by motors of reduced size [8].

The planetary gear system is widely used in robotics [9, 10], it is composed of one or more outer gears or planet gears, revolving around a sun gear driven by motor. Typically, the planets are mounted on a movable carrier, which rotates relatively to the sun gear. The planetary gearing systems may also include an outer ring gear or annulus, which meshes with the planet gears [9, 10, 11]. A planetary gear example modeled by SolidWorks (2014) is shown in Fig. 1. The advantages of planetary gears over parallel axis gears include high power density, large gear reduction in a small volume, multiple kinematic combinations, pure torsional reactions, and coaxial shafting. Among the disadvantages, one can note high bearing loads, inaccessibility and design complexity [9, 10].

The rotary inertia of the sun gear had a very significant influence on the dynamic behavior of planetary gear train [12]. The effect of gear dynamics and gear ratios on the inverse and forward dynamic models is a problem that is of current interest. For the inverse dynamics, the gear dynamics are most commonly approximated by a diagonal matrix added to the mass matrix [13, 14, 15]. Spong [16] explicitly assumes that the gyroscopic effects of the spinning rotor and gears are negligible. Springer et. Al [17] and Chen [18] used energy methods to add additional terms to the simple rigid model to account for the effects of gear ratios and gyroscopic forces. Walker and Orin [19] developed methods for the fast calculation of the manipulator mass matrix that could include the common gear approximation.

We model the actuator with planetary gear system by using SolidWorks (2020) with no backlash. The planetary gear serves to deliver high output torques. The sun-gear has as input the torque given by the motor and gives torque to the outer ring gear through planet-gears, the output of the outer ring gear connected with the successive link as shown Fig. 1.

In this paper, we are interested on the effects of the rotary inertia of the number of the equally spaced planet-gears on the dynamic of the robot and we will consider that the planet carrier is fixed. A two-revolute (2-R) planar robot is modeled using the SolidWorks as shown in Fig. 1. Motors on each joint coupled with a planetary gear system actuate the manipulator that holds an external mass on the end effector.

The rest of this paper is organized as follows: in the section 2 dynamic model of the manipulator, in section 3 the dynamic simulation, in section 4 discussion of the results followed by a conclusion in section 5.

Figure 1 The joint of robot's link actuated by planetary gear system

2. Dynamics modeling of robot actuated by (n) equally spaced planet-gears

2.1. Dynamics modeling of robot actuated by the 3 equally spaced planet-gears

2.1.1. Computation of Kinetic energy of robot

The total kinetic energy is given by the following relation : (See Fig. 2)

 $T=\sum_{i=0}^{\infty}$ $\binom{n}{i}$ $\binom{n}{i}$ $\binom{n}{i+1}$ $\binom{n}{i+1}$ $\binom{n}{i+1}$ $\binom{n}{i+1}$ $\binom{n}{i+1}$ $\binom{n}{i+1}$ $\binom{n}{i+1}$ $\binom{n}{i+1}$

It can be assumed that the contribution in the calculation of the kinetic energy of the (ring-gear) is included in that of the link on which the (ring-gear) is located, and thus the sole contribution of the sun-gear and planet-gears are to be computed. The sun-gears are located on the joint axes, the sun-gear (1) and planet-gears (1) are sitting on the ground and their weight will not affect the dynamics of the manipulators [5, 23].

The total kinetic energy contribution of Link (i) is given by:

T=T_(l_i)+T_(〖sg〗_(i+1)) +T_(〖pg2〗_(i+1))+2T_(〖pg1〗_(i+1)) …………. (2) T_(l_i)=1/2 m_(l_i) p ̇_(l_i)^T p ̇_(l_i) +1/2 ω_i^T I_i ω_i…………. (3) 〖 T〗_(〖sg〗_(i+1))=1/2 m_(〖sg〗_(i+1)) p ̇_(〖sg〗_(i+1))^T p ̇_(〖sg〗_(i+1)+) +1/2 ω_(〖sg〗_(i+1))^T I_(〖sg〗_i) ω_(〖sg〗_(i+1)) …………. (4) T_(〖pg1〗_(i+1))=1/2 m_(〖pg〗_(i+1)) p ̇_(〖pg〗_(1i+1))^T p ̇_(〖pg1〗_(i+1)+) +1/2 ω_(〖pg〗_(i+1))^T I_(〖pg〗_i) ω_(〖pg〗_(i+1)) …………. (5) T_(〖pg2〗_(i+1))=1/2 m_(〖pg〗_(i+1)) p ̇_(〖pg2〗_(i+1))^T p ̇_(〖pg2〗_(i+1)+)

 $+1/2 \omega$ (\lceil pg) $(i+1)$ [^]T I (\lceil pg) i) ω (\lceil pg) $(i+1)$) …………. (6)

 ϑ (\lceil (sg) \lceil (i+1)) and ϑ (\lceil (\lceil pg) \lceil (i+1)) are the angular position of the sun-gear and planet-gear respectively.

ϑ ̇_(〖sg〗_(i+1))=G_(sgr_(i+1)) ϑ ̇_i …………. (7) ϑ ̇_(〖pg〗_(i+1))=G_(pgr_(i+1)) ϑ ̇_i …………. (8)

here $G_{Sgr_{i}(i+1)}$ and $G_{Sgr_{i}(i+1)}$ are the gear reduction ratios.

In our case, the planet carrier is fixed, the gear reduction ratios can be calculated by using the relation of Willis [24, 25, 26]:

$$
G_{Sgr_{i}(i+1)} = G_{Sgr_{i}(i+1)} = Z_{rg}/Z_{sg} = \theta_{i} \quad \text{(sg)}_{i} \quad j/\theta_{i} \quad \text{(9)}
$$
\n
$$
G_{Sgr_{i}(i+1)} = G_{Sgr_{i}(i+1)} = Z_{rg}/Z_{pg} = \theta_{i} \quad \text{(9)}
$$

Where Z sgthe number of sun-gear teeth is, Z rg is the number of ring-gear teeth, and Z pg is the number of planet-gear teeth. The ring-gear rotates at the rate $G(Sgr_i(i+1))$ in the negative direction of the sun-gear because the sun gear and the ring gear rotate through a planet gear.

The angular velocity composition rule:

The rotation of the link, sun-gear and planet-gears is about the z axis:

$$
\omega_{-}(i+1)=\omega_{-}i+\omega_{-}(i,i+1)\ \dots\dots\dots\dots\dots(11)
$$

According to the angular velocity composition rule and the Eq. 7 the total angular velocity of the sun-gear is:

$$
\omega_{-}(\text{[sg]}_{-}(i+1)) = \omega_{-}i + G_{-}(sgr_{-}(i+1)) \vartheta_{-}i z_{-}(\text{[sg]}_{-}(i+1)) \dots \dots \dots \dots \dots (12)
$$

Where ω i is the angular velocity of link (i) on which the sun-gear is located and z_($\llbracket \text{sg} \rrbracket$ (i+1)) denotes the unit vector along the sun-gear axis.

z_(〖sg〗_(i+1))=z_(〖sg〗_i)=z_i=z_0=[0 0 1]^T…………. (13)

In similar fashion, the angular velocity of the planet gear:

$$
\omega \ ([\![pg]\!] \ (i+1) \) = \omega_i + G \ ([pgr_i(i+1) \) \ \vartheta \ i \ z \ ([\![pg]\!] \ (i+1) \) \ \dots \dots \dots \dots \ (14)
$$

where:

z $[$ $[$ $[$ $[$ $]$ $[$ $]$ $[$ $]$ $]$ $=$ $[$ $[$ $[$ $]$ $[$ $]$ $]$ $=$ $[$ $[$ $]$ $=$ $[$ $[$ $]$ $[$ $]$ $[$ $[$ $]$ $[$ $]$ $[$ $]$ $[$ $[$ $]$ $[$ $]$ $[$ $]$ $[$ $[$ $]$ $[$ $]$ $[$ $]$ $[$ $]$ $[$ $]$ $[$ $]$

Definition of the augmented link (i)

The augmented body (i) is defined as the fictitious body consisting of the particles of body (i) itself and of a mass, equal to that of bodies (i+1,i+2,...n) attached to $o_{1}(i+1)$. This definition can be applied regardless the type of joint (R) revolute joint or (p) prismatic joint) [28]. By using this definition, in our case, the mass of the augmented link (i) equals to the sum of the mass of link (i) with the mass of sun-gear and planet-gears in joint $(i+1)$ as shown Fig. 2 and the mass of link (i) and the mass of the external mass as shown Fig. 3.

In order to determine such parameters, it is worth associating the kinetic energy contributions of each sun-gear and planet-gears with those of the link on which it is located as shown Fig. 2. Hence, by considering the union of link (i)+sungear and planet-gears (i+1) (augmented Link(i)), the kinetic energy of the augmented link can be shown by using the following demonstrations. With reference to the center of mass of the augmented link, the linear velocities of the link, sun-gear and planet-gears can be expressed:

Figure 2 Characterization of augmented link (i) (link+sun-gear+planet-gears) for Kinetic energy

$$
p_{-}(l_{-}i) = p_{-}(C_{-}i) + \omega_{-}ixr_{-}(C_{-}i, l_{-}i) \dots (16)
$$
\n
$$
(p)_{-}(\text{Sg})_{-}(i+1) = p_{-}(C_{-}i) + \omega_{-}ixr_{-}(C_{-}i, \text{Sg})_{-}(i+1) \dots (17)
$$
\n
$$
p_{-}(\text{Sg1})_{-}(i+1) = p_{-}(\text{C}_-i) + \omega_{-}ixr_{-}(\text{C}_-i, \text{Sg1})_{-}(i+1) \dots (18)
$$
\n
$$
p_{-}(\text{Sg2})_{-}(i+1) = p_{-}(\text{C}_-i) + \omega_{-}ixr_{-}(\text{C}_-i, \text{Sg2})_{-}(i+1) \dots (19)
$$

Kinetic energy of link (i) relative to the overall centre of mass C_i

By substituting Eq. 16 into Eq. 3, the kinetic energy of link (i) is given by the following:

$$
T_{(l_i)} = 1/2 m_{(l_i)} p_{(c_i)}^T T p_{(c_i)} S(\omega_i) m_{(l_i)} r_{(c_i,l_i)} + 1/2 m_{(l_i)} \omega_i^T S^T (r_{(c_i,l_i)} S(r_{(c_i,l_i)}))
$$

\n
$$
\omega_i + 1/2 \omega_i^T T_i \omega_i
$$

By virtue of Steiner theorem, the following matrix is obtained:

I ̅_(l_i)=I_(l_i)+m_(l_i) S^T (r_(C_i,l_i))S(r_(C_i,l_i)) …………. (21)

where:

$$
\begin{bmatrix} 1 \end{bmatrix} \begin{bmatrix} 1 \end{bmatrix} = \begin{bmatrix} 1 \end{bmatrix} = \begin{bmatrix} 1 \end{bmatrix} \begin{bmatrix} 1 \end{bmatrix} \begin{bmatrix} 1 \end{bmatrix} \begin{bmatrix} 1 \end{bmatrix} = \begin{bmatrix} 1 \end{bmatrix} \begin{bmatrix} 1 \end{bmatrix} \begin{bmatrix} 1 \end{bmatrix} = \begin{bmatrix} 1 \end{bmatrix} = \begin{bmatrix} 1 \end{bmatrix} \begin{bmatrix} 1 \end{bmatrix} = \begin{bmatrix}
$$

and:

m_(l_i) r_(C_i,l_i)=[m_(l_i) r_(C_i,l_ix) m_(l_i) r_(C_i,l_iy) m_(l_i) r_(C_i,l_iz)]^T …………. (23)

S(r_(C_i,l_i))=[■(0&-r_(C_i,l_iz)&r_(C_i,l_iy)@r_(C_i,l_iz)&0&-r_(C_i,l_ix)@-r_(C_i,l_iy)&r_(C_i,l_ix)&0)] …………. (24)

 Γ _{Γ} $($ | \bot i) represents the inertia tensor of the link (i) relative to the overall centre of mass Γ Γ Γ \bot , Eq. 20 can be written as :

$$
T_{(l_i)} = 1/2 m_{(i_i)} p_{(c_i)}^T T p_{(c_i)} + p_{(c_i)}^T T S(\omega_i) m_{(l_i)}^T T (C_i) + 1/2 \omega_i^T T [l_i] \omega_i
$$

Kinetic energy of sun-gear relative to the overall center of mass $\lbrack \lbrack C \rbrack \rbrack$ i

By substituting Eq. 17 into Eq. 4 and exploiting Eq. 12, we obtain:

 $T_{-}([s]_{-}([i+1) - 1/2 m_{-}([s]_{-}([i+1) - 1/2 m_{-}([s]_{-}^{\circ}([i+1) - 1/2 m_{-}([s]_{-}^{\$ +1/2 ω_i^T Γ $[$ $[sg]$ $[$ (i+1)) ω_i+G_(sgr,i+1) q $[$ (i+1) z_($[sg]$ $[$ (i+1))^T I $[$ $[sg]$ $[$ (i+1) $]$ ω_i+1/2 $[$ G _sgr^2 $]$ $_{-}(i+1)$ q $_{-}(i+1)^2$ z_($[sg]_{-}(i+1)$)^T I_($[sg]_{-}(i+1)$) z_($[sg]_{-}(i+1)$) …………. (26)

By virtue of Steiner theorem, the following matrix is obtained:

$$
\Gamma_{-}(\text{Sg})_{-(i+1)} = I_{-}(\text{Sg})_{-(i+1)} + m_{-}(\text{Sg})_{-(i+1)} S^{\wedge}T(r_{-}(C_{-}i, \text{Sg})_{-(i+1)})S(r_{-}(C_{-}i, \text{Sg})_{-(i+1)}).
$$

Where:

$$
I_{(sg)}_{(i+1)(y)} = \begin{bmatrix} 0 & (1 - (1 - 0.5g) - (1 - 1.5g) - (1
$$

and:

m_(〖sg〗_(i+1)) r_(C_i,〖sg〗_(i+1))=[m_(〖sg〗_(i+1)) r_(C_i,〖sg〗_(i+1x)),m_(〖sg〗_i+1) r_(C_i,〖sg〗_(i+1y)) 〖 ,m〗_(〖sg〗_(i+1)) r_(C_i,〖sg〗_(i+1z))]^T …………. (29)

S(r_(C_i,〖sg1〗_(i+1)))=[■(0&-r_(C_i,〖sg〗_(i+1z))&r_(C_i,〖sg〗_(i+1y))@r_(C_i,〖sg〗_(i+1z))&0&-r_(C_i,〖sg 〗_(i+1x))@-r_(C_i,〖sg〗_(i+1y))&r_(C_i,〖sg〗_(i+1x))&0)] …………. (30)

 Γ (\lceil \lceil \lceil \lceil \lceil \lceil \rceil))represents the inertia tensor of the sun-gear (i+1) relative to the overall center of mass \lceil \lceil \lceil \rceil i.

Kinetic energy of planet-gear (1) relative to the overall center of mass C_i

By substituting Eq. 18 into Eq. 5 and exploiting Eq. 14, we obtain

 $T_{-}([\text{pg1}]_{-}(\text{i}+1)] = 1/2 \text{ m}[(\text{pg2}]_{-}(\text{i}+1)] \text{ p}[(\text{c}_i)^{\wedge}T \text{ p}[(\text{c}_i) + \text{p}[(\text{c}_i)^{\wedge}T \text{ S}(\omega_i)] \text{ m}[(\text{g2}]_{-}(\text{i}+1)] \text{ r}[(\text{c}_i, [\text{pg1}]_{-}(\text{c}_i)]$ $_{(i+1)} + 1/2$ ω_i^T Γ $[$ $[pg]$ $[$ $(i+1)$) ω_i+G_(pgr,i+1) q $_{(i+1)}$ z_($[pg]$ $[$ $(i+1)$)^T I_($[pg]$ $[$ $(i+1)$) ω_i+1/2 $[$ G_pgr^2] $[(i+1) q - (i+1)^2 z - (lpg)] - (i+1) q - (i$

By virtue of Steiner theorem, the following matrix is obtained:

 Γ $[$ $[pg1]$ $[$ i+1) $]=$ $[$ $[pg1]$ $[$ i+1) $]+$ m $[$ $[pg]$ $[$ i+1) $)$ $S^{\wedge}T$ (r $[$ C_i , $[pg1]$ $[$ $[$ $]$ $[$ $[$ $]$ $[$ $[$ $]$ $[$ $]$ $[$ $]$ $[$ $]$ $[$ $]$ $[$ $]$ $[$ $]$ $[$ $]$ $[$ $]$ $[$ $]$ $[$ $]$ $[$ $]$ $[$ $]$ $[$ $]$ $[$ $]$ $[$ $]$ $[$

where:

 $I_{\llcorner}([pg1]_{\llcorner}(i+1))=[■(I_{\llcorner}([pg1]_{\llcorner}(i+1(xx))] & .(i+1(xy) - (i+1(xy)))& I_{\llcorner}([pg1]_{\llcorner}(i+1(xx))] & .(i+1(yx))$)&I_(〖pg1〗_(i+1(yy)))&I_(〖pg1〗_(i+1(yz)))@I_(〖pg1〗_(i+1(zx)))&I_(〖pg1〗_(i+1(zy)))&I_(〖pg1〗 $[(i+1(zz))$))] …………. (33)

and:

 m_{-} ($\lceil \log_{2} \rceil_{i+1}$) r $_{-}$ (C_i, $\lceil \log_{2} \rceil_{i+1}$) = $\lceil m_{-} \rceil_{i}$ $\lceil \log_{2} \rceil_{i+1}$) r $_{-}$ (C_i, $\lceil \log_{2} \rceil_{i+1}$, $\lceil \log_{2} \rceil_{i+1}$) r $_{-}$ (C_i, $\lceil \log_{2} \rceil_{i+1}$) r $_{-}$ (C_i, $\lceil \log_{2} \rceil_{i+1}$) r $_{-}$ (〗_(i+1y)) 〖 ,m〗_(〖pg〗_(i+1)) r_(C_i,〖pg1〗_(i+1z))]^T …………. (34)

 $S(r_{c}(C_i, \mathbb{F}pg1)_{c}(i+1))$) = $[\blacksquare(0&r_{c}(C_i, \mathbb{F}pg1)_{c}(i+1z)]$ $\&r_{c}(C_i, \mathbb{F}pg1)_{c}(i+1y)]$ \emptyset $r_{c}(C_i, \mathbb{F}pg1)_{c}(i+1z)$ $\&0&r_{c}(C_i, \mathbb{F}gg1)$ $[pg1]$ $(i+1x)$ $[@-r_C(C_i, [pg1] - (i+1y)] &c_C(C_i, [pg1] - (i+1x)] &0]$ …………. (35)

 $\Gamma_{\text{L}}(\llbracket \text{pg1} \rrbracket_{\text{L}})$ represents the inertia tensor of the planet-gear relative to the overall center of mass $\llbracket \text{C} \rrbracket_{\text{L}}$.

Kinetic energy of planet-gear (2) relative to the overall center of mass C_i

By substituting Eq. 19 into Eq. 6 and exploiting Eq. 14, we obtain

 T_{ℓ} ($\lceil \log 2 \rceil$ _{_(i+1)})=1/2 m_{_}($\lceil \log 2 \rceil$ ₁ (i+1)) p $\lceil (c_i)^{AT} p \rceil$ ₁ $\lceil (c_i)^{AT} S(\omega_i)$ m_{_}($\lceil \log 2 \rceil$ ₁ $\lceil (c_i)^{LT} p \rceil$ ₁ $\lceil (c_i)^{AT} S(\omega_i)$ m_{_(} $\lceil (c_i)^{T} S(\omega_i) \rceil$ $_{-}$ (i+1)) +1/2 ω_i^T Γ $\left[$ $\left[$ $\right[pg2]$ $_{-}$ $(i+1)$) ω_i+G_(pgr,i+1) q $_{-}$ (i+1) z_($\left[$ $\left[pg2\right]$ $_{-}$ $(i+1)$) $\left[$ $\left[pg2\right]$ $_{-}$ $(i+1)$ $\right]$ ω $_{-}$ i+1/2 $\left[$ G_pgr^2] $[(i+1) q]_{(i+1)^2 z}$ $[(i+2) q]_{(i+1)^2 z}$ $[(i+1) q]_{(i+1) z}$ $[(i+1) q]_{(i+$

By virtue of Steiner theorem, the following matrix is obtained:

$$
\Gamma_{-}(\text{[[pg2] _-(i+1))] = I_{-}(\text{[[pg2] _-(i+1)}) + m_{-}(\text{[[pg] _-(i+1)) S^T(r_{-}(C_i, \text{[[pg2] _-(i+1))}) S(r_{-}(C_i, \text{[[pg2] _-(i+1))})
$$

where:

$$
I_{(pg2)}_{(i+1)} = [m(I_{(pg2)}_{(i+1(xy))})&I_{(pg2)}_{(i+1(xy))})&I_{(pg2)}_{(i+1(xx))})&I_{(g2)}_{(i+1(xx))})&I_{(g2)}_{(i+1(xy))})&I_{(g2)}_{(i+1(xy))})&I_{(g2)}_{(i+1(xy))})&I_{(g2)}_{(i+1(xx))}&I_{(g2)}_{(i+1(xy))})&I_{(g2)}_{(i+1(xx))}&I_{(g2)}_{(i+1(xx
$$

and:

```
m_{\ell} [pg]_{\ell} (i+1) r_{\ell} C_{\ell}, [pg2]_{\ell} (i+1) =[m_{\ell} [pg]_{\ell} (i+1) r_{\ell} C_{\ell}, [pg2]_{\ell} (i+1x) [gh]_{\ell} [hg]_{\ell} [i+1) r_{\ell} (C_{\ell}, [gh]_{\ell}pg2〗_(i+1y) ) 〖 ,m〗_(〖pg〗_(i+1) ) r_(C_i,〖pg2〗_(i+1z) ) ]^T…………. (39)
```
S(r_(C_i,〖pg2〗_(i+1)))=[■(0&-r_(C_i,〖pg2〗_(i+1z))&r_(C_i,〖pg2〗_(i+1y))@r_(C_i,〖pg2〗_(i+1z))&0&-r_(C_i, $[pg2]$ $[i+1x]$ $@-r(C_i, [pg2]$ $[i+1y]$ $@r(C_i, [pg2]$ $[i+1x]$ $@0]$ …………. (40)

 Γ ($\lceil \log 2 \rceil$ (i+1))represents the inertia tensor of the planet-gear relative to the overall center of mass $\lceil \zeta \rceil$ i.

Summing the contributions in Eq. 25, 26, 31 and Eq. 36 as in Eq. 2 gives the expression of the kinetic energy of augmented link (i) in the form :

T_i=1/2 m_i p _(c_i)^T p _(c_i)+1/2 ω_i^T Γ _i ω_i+G_(sgr,i+1) q _(i+1) z_(〖sg〗_(i+1))^T I_(〖sg〗_(i+1)) ω_i+1/2 〖 G_Sgr^2] $(j+1)$ $q (i+1)^2 z ($ $[sg] (i+1))^T I ($ $[sg] (i+1) z ($ $[sg] (i+1) +G(pgr,i+1) q (i+1) z ($ $[pg]$ $_{(i+1)})$ ^T $_{(s+1)}$ $_{(s+1)}$ $_{(s+1)}$ $_{(s+1)/2}$ $_{(s+1)/$ 〗_(i+1))+2G_(pgr,i+1) q ̇_(i+1) z_(〖pg2〗_(i+1))^T I_(〖pg1〗_(i+1)) ω_i+〖G_pgr^2〗_(,i+1) q ̇_(i+1)^2 z_(〖pg1 $\left[\right]_{-}$ (i+1))^T I_($\left[\right]_{p}$ g1 $\left[\right]_{-}$ (i+1)) z_($\left[\right]_{p}$ g1 $\left[\right]_{-}$ (i+1)) …………. (41)

where $m_i = m_l[ln] + m_l[$ $[sg]_{-i+1}$)+3m₋($[pg]_{-i+1}$) and $l_i = l_l[ln] + l_l[$ $[sg]_{-i+1}$)+2 $l_i[ng1]_{-i+1}$)+ $l_i[$ 〖pg2〗_(i+1))are respectively the overall mass and inertia tensor relative to the position of center of mass of the augmented link.

The inertia tensor of the augmented link can be written as:

I ̅_i=[■(I ̅_ix&-I ̅_ixy&-I ̅_ixz@-I ̅_iyx&I ̅_iyy&-I ̅_iyz@-I ̅_izx&-I ̅_izy&I ̅_izz)] …………. (42)

The position of center of mass of augmented link can be expressed by:

m_(l_i) P_(l_i)+m_(〖sg〗_(i+1)) P_(〖sg〗_(i+1))+2m_(〖pg〗_(i+1)) P_(〖pg1〗_(i+1))+m_(〖pg〗_(i+1)) P_(〖 pg2〗_(i+1))=m_i P_(C_i) …………. (43)

Computing the inertia tensor of the augmented link (link+external mass) relative to the position of center of mass as shown Fig. 3:

Figure 3 Characterization of augmented link (i) (link+external mass) for Kinetic energy

The external mass is a sphere with mass $(m_{e}m)$ and radius (r). The inertia tensor about its center of mass is :

 $\lbrack \!\lbrack \rbrack \!\rbrack \!\rbrack$ $[(e_m x x) \& \lbrack \!\lbrack (e_m x y) \& \lbrack \!\lbrack (e_m x z) \& \lbrack \!\lbrack (e_m x y) \& \lbrack \!\lbrack (e_m y z) \& \lbrack \!\lbrack (e_m x z) \& \lbrack \!\lbrack (e_m x x) \& \lbrack \!\lbrack (e_m x y) \& \lbrack \!\lbrack (e_m x y) \& \lbrack \!\lbrack (e_m x z) \& \lbrack \!\lbrack (e_m x y) \& \lbrack \!\lbrack (e_m x y) \$)&I_(e_mzz))] (44)

 $m_{\text{e}}(e_m)$ r_{_}(C_i,e_m)=[m_(e_m) r_(C_i,e_mx),m_(e_m) r_(C_i,e_my) ,m_(e_m) r_(C_i,e_mz)]^T …………. (45)

 $(r_{\text{L}}(C_i,e_m))$ = $[\blacksquare(0&r_{\text{L}}(C_i,e_mz)&r_{\text{L}}(C_i,e_my)@r_{\text{L}}(C_i,e_mz)&0&r_{\text{L}}(C_i,e_mx)@r_{\text{L}}(C_i,e_mx)&0]$] (46)

 Γ [[](e_m)=I[[](e_m)+m[[](e_m) S^T (r[[](C_i,e_m))S(r[[](C_i,e_m)) …………. (47)

where $\Gamma_{\text{e}}(e_{\text{m}})$ represents the inertia tensor of the external mass relative to the overall centre of mass $\lbrack\!\lbrack$ C $\lbrack\!\rbrack$ \lbrack .

The position of center of mass of augmented link can be expressed by:

$$
m_{l,i} p_{l,i} + m_{e,m} p_{e,m} = m_{i0} p_{l,i} + m_{i1} p_{i1}
$$

The kinetic energy of the augmented link can be expressed by:

 $T_i=1/2$ m_i0 p $(C_i)^T$ p $(C_i + 1/2$ ω_i[^]T Γ ₁₀ ω_i …………. (49)

where $\lbrack \lbrack m \rbrack \rbrack$ \lbrack \lbrack relative to the position of center of mass of the augmented link.

On the assumption that the link, sun-gear, planet-gears and external mass have a symmetric mass distribution about the axis of rotation and the axis of link (i) frame coincides with the central axis of

inertia, then the inertia products are null and the inertia tensor relative to the center of mass is a diagonal matrix [5, 23].

The inertia tensor of sun-gear about its center of mass:

I_(〖sg〗_(i+1))^(〖sg〗_(i+1))=[■(I_(〖sg〗_(i+1(xx)))^(〖sg〗_(i+1))&0&0@0&I_(〖sg〗_(i+1(yy)))^(〖sg〗 _(i+1))&0@0&0&I_(〖sg〗_(i+1(zz)))^(〖sg〗_(i+1)))] …………. (50)

The inertia tensor of planet-gear about its center of mass:

$$
I_{(ng)}_{(i+1)})^{\wedge}(\ [pg]_{(i+1)}) = [m(I_{(ng)}_{(i+1(xx))})^{\wedge}(\ [pg]_{(i+1)})\&0\&0\&0\&1(\ [pg]_{(i+1)(y)}^{\wedge})^{\wedge}(\ [pg]_{(i+1)(zx))^{\wedge}(\ [pg]_{(i+1)(y)}^{\wedge})^{\wedge}(\ [pg]_{(i+1)(zx))^{\wedge}(\ [pg]_{(i+1)(zx))^{\wedge}(\ [ng]_{(i+1)(zx))^{\wedge}(\ [ng]_{(i+1)(zx))^{\
$$

Since the aim is to determine a set of dynamic parameters independent of the manipulator joint configuration, it is worth referring the inertia tensor of the link Γ ₋i to frame R₋i attached to the link and the inertia tensor of sun-gear and planetgear I_($\lceil \{sg\} \rceil$ _(i+1)),I_($\lceil \{pg\} \rceil$ _(i+1)) to frame R_($\lceil \{sg\} \rceil$ _(i+1)) andR_($\lceil \{pg\} \rceil$ _(i+1)) respectively,so that it is diagonal. In view of Eq. 50 and Eq. 51 one has:

 $I_{\{s,g\}$ $_{[i+1)}$ $_{z}$ $_{[s,g]}$ $_{[i+1)}$ $_{=}$ $R_{\{s,g\}$ $_{[i+1)}$ $_{[s,g]}$ $_{[i+1)}$ $_{(s,g)}$ $_{[i+1)}$ $_{R_{\{s,g\}$ $_{[i+1)}$ $_{T}$ $_{z}$ $_{[s,g]}$ $_{[i+1)}$)=I $_{[s]}$ $_{[s]}$ $_{[i+1)}$ $_{[s]}$ $_{[s]}$ $_{[i+1]}$ $_{[i+1]}$ $_{[i+1]}$ $_{[i+1]}$ $_{[s]}$

I_(〖pg〗_(i+1)) z_(〖pg〗_(i+1))=R_(〖pg〗_(i+1)) I_(〖pg〗_(i+1))^(〖pg〗_(i+1)) R_(〖pg〗_(i+1))^T z_(〖pg〗 $[(i+1)] = [[(\text{pg}]\cdot(i+1)] \cdot z[(\text{pg}]\cdot(i+1)] \cdot ... \cdot (53)$

where $I_{\{s\}}(sg)$ $(i+1)$)= $I_{\{s\}}(sg)$ $(i+1(zz))$ and $I_{\{s\}}(sg)$ $(i+1)$)= $I_{\{s\}}(sg)$ $(i+1(zz))$)= $I_{\{s\}}(sg)$ $(i+1(zz))$)= $I_{\{s\}}(sg)$ $\lceil \text{pg2} \rceil$ (i+1(zz))) denote the constant scalar moment of inertia of the sun-gear and planet-gear about the rotation axis.

Therefore, the kinetic energy shown in Eq. 41 becomes:

 $T_i=1/2$ m_i p $(c_i)^(i^{\wedge}T)$ p $(c_i)^{i+1/2}$ ω_i^T T_i^i i ω_i+G_(sgr,i+1) q $(i+1)$ I_(〖sg〗_(i+1)) z_(〖sg〗_(i+1))^(i^T) $\omega_i+1/2$ ζ_s ζ_s ζ_i+1) $q^-(i+1)^2$ I_{ζ_s} ζ_s I_{i+1}) +3G_(pgr,i+1) $q^-(i+1)$ I_{ζ_s} ζ_s I_{i+1} I_{i+1} I_{i+1} $_{(i+1)})$ ^(i^T) $\omega_{i+3/2}$ $\left[\right[G_{pgr}^2 \right]_{(i+1)}$ q' $_{(i+1)^2}$ $I_{(s+1)}$ $\left[\right[\left[\right]$ $_{(i+1)}$) …………. (54)

According to the linear velocity composition rule for link (i), one may write:

 p^{\prime} $(c_i)^{\prime}$ i=p'_i^i+ω_i^i×r_(i, C_i)^i …………. (55)

where all the vectors have been referred to frame (i), note that r (i,C i)^iis fixed in such a frame. Substituting Eq. 55 in Eq. 54 gives:

 $T_i=1/2$ m_i p'_i^(i^T) p'_i^i+p'_i^(i^T) $S(\omega_i^i)$ m_i r_(i,C_i)^i +1/2 ω_i^i (i^T) $\overline{\Gamma_i}$ i^i ω_i^i +G_(sgr,i+1) q'_(i+1) I_(sg] $_{(i+1)}$) z $_{(s,g)}$ $_{(i+1)}$ ^{\wedge}(i^{\wedge}T) $\omega_{i+1/2}$ (G_{sgr}^2) $_{(i+1)}$ q $_{(i+1)}^2$ $(G_{(i+1)}^2)$ $(G_{(i+1)}^2)$ $(G_{(i+1)}^2)$ $(G_{(i+1)}^2)$ $(G_{(i+1)}^2)$ $(G_{(i+1)}^2)$ $(G_{(i+1)}^2)$ $(G_{(i+1)}^2)$ $(G_{(i+1)}^2)$ $\lceil \pi \rceil$ $\lceil \log \rceil$ $\lceil \frac{\lceil \log \rceil}{\lceil \log \rceil}$ $\lceil \frac{\lceil \frac{\lceil \log \rceil}{\lceil \log \rceil}}{\lceil \frac{\lceil \log \rceil}{\lceil \log \rceil}}$ $\lceil \frac{\lceil \frac{\lceil \log \rceil}{\lceil \log \rceil}}{\lceil \frac{\lceil \log \rceil}{\lceil \log \rceil}}$ $\lceil \frac{\lceil \log \rceil}{\lceil \log \rceil}$ $\lceil \frac{\lceil \log \rceil}{\lceil \log \rceil}$ $\lceil \frac{\lceil \log \rceil}{$

In similar fashion, the kinetic energy of the augmented link (Link+sun-gear+planet-gears) actuated by (n) equally planet-gears:

 $T_i=1/2$ m_i p'_i^(i^T) p'_i^i+p'_i^(i^T) $S(\omega_i^i)$ m_i r_(i,C_i)^i +1/2 ω_i^i (i^T) $\overline{\Gamma_i}$ i^i ω_i^i i+G_(sgr,i+1) q'_(i+1) I_(s_g] $_{(i+1)}$ $_{z}$ $_{(s,g)}$ $_{(i+1)}$ $^{(i+1)}$ $_{(\iota+1/2)}$ $_{(s, s)^2}$ $_{(i+1)$ q $_{(i+1)}$ $^{(i+1)2}$ $_{(i+1)$ $(s, s)}$ $_{(i+1)$ $(s, s)}$ $_{(i+1)$ $+(n)$ $_{(s, s)}$ $_{(s+1)$ q $_{(i+1)}$ $I(\lceil p_g \rceil_{(i+1)})$ z $(\lceil p_g \rceil_{(i+1)})^{\wedge}(i^{\wedge}T)$ ω i+ $(\lceil n \rceil)/2$ $\lceil G_p \rceil_{(i+1)}$ q' $(i+1)^{\wedge}2$ $\lceil (\lceil p_g \rceil_{(i+1)})$ …………. (57)

where

$$
\Gamma_i^i = \Gamma_i^i + m_i^i S^T (r_i, C_i^i)^i S (r_i, C_i^i)^i
$$
 (58)

represents the inertia tensor with respect to the origin of frame (i), according to Steiner theorem. Let:

$$
r_{i}(i,C_i)^{i} = [r_{i}(i,C_ix)^{i} \quad [r_{i}(i,C_i)y)^{i} \quad [r_{i}(i,C_i) \quad [i,C_i]^{i}]^{i}
$$

The first moment of inertia is:

$$
m_i r_{i,c_i}^j \cap i = [m_i r]_{i,c_i}^j \cap i [m_i r]_{i,c_i}^j \cap m_i r]_{i,c_i}^j \cap i [m_i r]_{i,c_i}^j \cap j [m_i r]_{i,c_i}^j \cap m_i m_i (60)
$$

From Eq. 58 the inertia tensor of augmented link (i) is:

 Γ_i^i i=[$\Pi(\Gamma_i^j$ [I Γ_i^j (r_(s_(i+1),C_iy)^(i^2)+r_(s_(i+1),C_iz)^(i^2))&- Γ_i^j [ixy^i m_i r_(s_(i+1),C_ix)^i $r_{s}(s_{i-1},C_{i})^{\wedge}i\&-1$ $\exists x^{\wedge}i$ m_i $r_{s}(s_{i-1},C_{i})^{\wedge}i$ $\exists r_{s}(s_{i-1},C_{i-1})^{\wedge}i\&-1$ $\exists x^{\wedge}i$ m_i $r_{s}(s_{i-1},C_{i-1})^{\wedge}i$ $r_{s_{i}}(s_{i+1}),C_{i}y^i\rightarrow s_{i+1}$ (r_(s_(i+1),C_ix)^(i^2)+r_(s_(i+1),C_iz)^(i^2))&-I _iyz^i m_i r_(s_(i+1),C_iy)^i $r_{s}(s_{i+1}),C_{i}z^i$ ['] i @- Γ_{i} ixz^i m_i $r_{s}(s_{i+1}),C_{i}x^i$ ' i $r_{s}(s_{i+1}),C_{i}z^i$ ' i &- Γ_{i} iyz^i m_i $r_{s}(s_{i+1}),C_{i}y^i$ ' i r_{s} (i+1),C_iz)^i&I_izz^i+m_i (r_(s_(i+1),C_ix)^(i^2)+r_(s_(i+1),C_iy)^(i^2)))] …………. (61)

 Γ i^i= $[\blacksquare(\Gamma \text{ i}xx^i\&\neg \Gamma \text{ i}xy^i\&\neg \Gamma \text{ i}xz^i\&\neg \Gamma \text{ i}xy^i\&\neg \Gamma \text{ i}xy^i\&\neg \Gamma \text{ i}xz^i\&\neg \$

In similar fashion, the kinetic energy of the augmented link (link+external mass) actuated by (n) equally spaced planetgears:

 $T_i=1/2$ m_io p _i^(i^T) p _i^i+p _i^(i^T) $S(\omega_i^i)$ m_io r_(e_m,C_i)^i +1/2 $\omega_i^i(i^T)$ $\overline{\Gamma}_i$ io^io $\omega_i^i(i^+ + G_i$ (sgr,i+1) $q^-(i+1) I^((sg)^-(i+1)) z^((sg)^-(i+1))^(i^{\wedge}T) \omega_i+1/2 [(G_sgr^{\wedge}2)]^-(i+1) q^-(i+1)^{\wedge}2 I^((sg)^-(i+1))$)+(n)G_(pgr,i+1) q ̇_(i+1) I_(〖pg〗_(i+1)) z_(〖pg〗_(i+1))^(i^T) ω_i+((n))/2 〖G_pgr^2〗_(,i+1) q ̇_(i+1)^2 I_(〖pg $\left[\right]$ (i+1) $\right]$ …………. (63)

where

$$
\Gamma_i^{\text{in}} = \Gamma_i^{\text{in}} \cdot \text{in} + \text{min} \, \text{S}^T \, \text{tr} \, (\text{r}_{\text{in}} \, \text{m}_{\text{out}} \, \text{in} \, \text{S}^T \, \text{tr} \, (\text{e}_{\text{in}} \, \text{m}_{\text{out}} \, \text{in} \, \text
$$

represents the inertia tensor with respect to the origin of frame (i), according to Steiner theorem. Let:

 r_{e_m,C_i}^{max} r^{(e_m,C_i)^i=[r_(e_m,C_ix)^i,r_(e_m,C_iy)^i,r_(e_m,C_iz)^i]^T …………. (65)}

The first moment of inertia is:

 m_io r_(e_m,C_i)^i=[$\lbrack \lbrack m_io$ r] $\lbrack (e_m, C_ix)$ ^i $\lbrack \lbrack \lbrack m \rbrack$ $_io$ r] $\lbrack (e_m, C_iy)$ ^i $\lbrack \lbrack \lbrack m \rbrack$ $_io$ r] $\lbrack (e_m, C_iz)$ ^i]^T ………….. (66)

 $\bar{\Gamma}$ io^io= $[\blacksquare(\Gamma \text{ iox}x^{\wedge}i\sigma+m\text{ i}o \text{ (}r(\Gamma \text{ e}_m,C\text{ i}y)^{\wedge}i\sigma^2 +r(\Gamma \text{ e}_m,C\text{ i}z)^{\wedge}i\sigma^2)]$ &- Γ ioxy^i m_io r_(e_m,C_ix)^i r_(e_m,C_iy)^i&- Γ joxz^i m_io r_(e_m,C_ix)^i r_(e_m,C_iz)^i@- Γ _ioxy^i m_io r_(e_m,C_ix)^i r_(e_m,C_iy)^i& Γ _ioyy^io+m_io (r_(e_m,C_ix)^(i^2)+r_(e_m,C_iz)^(i^2))&-I ̅_ioyz^i m_io r_(e_m,C_iy)^i r_(e_m,C_iz)^i@-I ̅_ioxz^i m_io r_(e_m,C_ix)^i $r_{\rm L}(e_m, C_1; z)$ [']i&- $\Gamma_{\rm L}$ ioyz^i m_io r_(e_m,C_iy)^i r_(e_m,C_iz)^i& $\Gamma_{\rm L}$ iozz^io+m_io (r_(e_m,C_ix)^(i^2)+r_(e_m,C_iy)^(i^2)))] …………. (67)

 Γ_i io^io= $[\blacksquare(\Gamma_i$ ioxx^io&- Γ_i ioxy^io& Γ_i ioxz^io@- Γ_i ioxy^io& Γ_i ioyy^io& $\blacksquare(\Gamma_i)$ _ioyz^io@- Γ_i iozx^io&- \lceil _iozy^io& \lceil _iozz^io)] …………. (68)

2.1.2. Newton–Euler Formulation

The Newton–Euler formulation describes the motion of the link in terms of a balance of forces and moments acting on it [27, 29], as shown Fig. 4.

Figure 4 Characterization of augmented link (i) for Newton-Euler formulation

The Newton equation for the translational motion of the center of mass can be written as:

f_i-f_(i-1)+m_i g_0=m_i p (C_i) …………. (69)

The Euler equation for the rotational motion of the link (referring moments to the center of mass) can be written as:

$$
\mu_i + f_i \times r_i(i-1, C_i) - \mu_i(i+1) - f_i(i+1) \times r_i(i, C_i) = d/dt (r_i \omega_i + G_i(sgr, i+1) q_i(i+1) I_{\{sgr\}} - (i+1) z_i(sgr, i+1) q_i(i+1) - 3G_i(sgr, i+1) q_i(i+1) I_{\{sgr\}} - (i+1) z_i(sgr, i+1))
$$

WhereG_(sgr,i+1) q _(i+1) I_(〖sg〗_(i+1)) z_(〖sg〗_(i+1))and G_(pgr,i+1) q _(i+1) I_(〖pg〗_(i+1)) z_(〖pg〗_(i+1)) are the angular momentum of the sun-gear and planet-gear respectively. Notice that the gravitational force m_i g₀ does not generate any moment, since it is concentrated at the centre of mass.

In our case, the angular momentum of the carrier equals to zero, because the carrier is fixed.

The inertia tensor of the augmented link (i) in the base: Γ _i=R_i (I_i^i) Γ R_i^T where R_i represents the rotation matrix from frame (i) to the base frame. Substituting this relation in the first term on the right-hand side of Eq. 70 yields:

 d/dt $(\Gamma_i \omega_i) = R_i$ $[\Pi_i^i \Gamma_i^i]$ $[\Pi_i^i \Gamma_i \omega_i + R_i$ $[\Pi_i^i \Gamma_i^i]$ $[\Pi_i^i \Gamma_i \omega_i]$ $[\Pi_i^i \Gamma_i^i]$ $[\$ $ω_i+R_i$ (I_i^i) R_i^i $T Sⁱ$ $T ω_i$ $)$ $ω_i+R_i$ (I_i^i) R_i^i $T ω_i$ $i = I_i ω_i + ω_i$ $s(I_i^i ω_i)$ …………. (71)

where:

 $ω$ i= [[ω ix,ω iv [,ω] iz]] ^T…………. (72)

In our case, the rotation is done only in (z)axis, Eq. 72 becomes:

 $ω_i = [(0 0 ω_i z_i)]^N$ (T) …………. (73)

and:

 $S(\omega_i)$ = [$\blacksquare(0\&\omega_iz\&0\&\omega_iz\&0\&0\&0\&0\&0\&0\]$ (74)

The second term of Eq. 71 represents the gyroscopic torque induced by the dependence of Γ_{\perp} on link orientation. Moreover, by observing that the unit vector z_{I} $[sg]$ $(i+1)$) rotates accordingly to link(i), the derivative needed in the second term on the right-hand side of Eq. 70 is:

 d/dt (G_(sgr,i+1) q _(i+1) I_(〖sg〗_(i+1)) z_(〖sg〗_(i+1))+G_(sg,i+1) q _(i+1) I_(〖pg〗_(i+1)) z_(〖pg〗_(i+1)) $)=G_{s}(\text{sgr},i+1)$ $(q^{n}(i+1)$ I_{s} $[$ $[sg]_{s}$ $[$ $(i+1)$ $]$ z $[$ $[sg]_{s}$ $[$ $(i+1)$ $]$ $[$ $[sg]_{s}$ $[$ $[$ $j+1)$ $[$ $[sg]_{s}$ $[$ $(i+1)$ $]$)+G_(pgr,i+1) (q ̆_(i+1) I_($[\wp g]$ _(i+1)) z_($[\wp g]$ _(i+1))+q _(i+1) I_($[\wp g]$ _(i+1)) ω _i×z_($[\wp g]$ _(i+1))) ………… (75)

In deriving Eq.71, the operator S has been introduced to compute the derivative of R_i, also the property S^T (ω _i) ω i=0 has been utilized.

By substituting Eq. 71, 75 in Eq. 70, the resulting Euler equation is:

 μ_i i+f_i×r_(i-1,C_i)- μ_i (i+1)-f_(i+1)×r_(i,C_i)= Γ_i ω $_i$ i+ω_i×(Γ_i i ω_i)+G_(sgr,i+1) (q $(i+1)$ I_(〖sg〗(i+1)) z_(〖sg〗 $_{(i+1)}$)+q $_{(i+1)}$ $_{(i)}$ $_{s}$ $_{(i+1)}$ $_{(i+1)}$ $_{(i+2)}$ $_{(i+1)}$ $_{(i+1)}$ $_{-}(i+1)$)+q $_{-}(i+1)$ $_{-}$ $[$ $\lceil \log \rceil$ $_{-}(i+1)$ \rceil ω $_{-}i \times z$ $[$ $\lceil \log \rceil$ $_{-}(i+1)$ \rceil \rceil (76)

The generalized force at Joint (i) can be computed by projecting the moment μ_i for a revolute joint, along the joint axis. In addition, there is the contribution of the inertia torques $G_{\text{Sgr},i}$ $I_{\text{Sgr},i}$ $I_{\text{Sgr},i}$ \in \mathbb{Z} $S_{\text{Sgr},i}$ \in \mathbb{Z} $S_{\text{Sgr},i}$ \in \mathbb{Z} $S_{\text{Sgr},i}$ \in \mathbb{Z} $S_{\text{Sgr},i}$ \in \mathbb{Z} $G_{\text{L}}(pgr,i) I_{\text{L}}([\text{L}pg][\text{L}j] \omega_{\text{L}}([\text{L}pg][\text{L}j]^{\wedge}T z_{\text{L}}([\text{L}pg][\text{L}j])$ of the sun-gear and planet-gears respectively.

he torque exerted at Joint (i) is expressed by:

τ_i=μ_i^T z_(i-1)+G_(sgr,i) I_(〖sg〗_i) ω ̇_(〖sg〗_i)^T z_(〖sg〗_i)+3G_(pgr,i) I_(〖pg〗_i) ω ̇_(〖pg〗_i)^T z_(〖pg 〗_i)+F_vi ϑ ̇_i+F_si sgn(ϑ ̇_i) …………. (77)

Where joint viscous torque (F_vi) and F_si Coulomb friction torque have been included, and in our case, we consider them neglected.

The N-E recursive

The following equations are calculated in current frame, because the recursion is computationally more efficient if all vectors are referred to the current frame on link (i) [27].

• The initial condition

$$
\omega_0^0
$$
 = ω_0^0 = 0^0 = p_0^0 = 0^0 = 0 \dots (78)

 p^{th} 0^{th} $0 = [0 \text{ g } 0]$ ^{\wedge} (T) … [79]

Forward recursive equations in current frame

ω i^i=R_i^(〖i-1〗^T) (ω_(i-1)^(i-1)+θ _i z_0) …………. (80)

ω $i^i=R i^{\prime}$ [[i-1] ^T) (ω $(i-1)^{i-1}+θ$ i z $(i-1)+θ$ i ω $(i-1)^{i-1} \times z$ 0) …………. (81)

ω ̇_(〖sg〗_i)^(i-1)=(ω ̇_(i-1)^(i-1)+G_sgri q ̈_i z_(〖sg〗_i)^(i-1)+G_sgri q ̇_i ω_(i-1)^(i-1)×z_(〖sg〗_i)^(i-1)) …………. (82)

 $ω$ $[$ $[$ $[pg]$ $]$ $[)$ ^(i-1)=(ω $[$ i-1)^(i-1)+G $[$ (pgr,i) q i $[$ i z $[$ (g $]$)^(i-1)+G $[$ gri q i ω $[$ i $|$ $]$ $[$ $($ i-1) \times z $[$ (g $]$ $]$ $($ i-1) $]$ …………. (83)

 p^i $i^i = R_i^i$ $[1, 1]$ $[1, 1$

 p^{\dagger} $(c_i)^i$ = p i^i i + ω i^i i × $r_{i}(i, C_i)^i$ + ω i^i i × $(\omega_i i^i x r_{i}(i, C_i)^i)$ …………. (85)

Backward recursive equations in current frame

f_i^i=R_(i+1)^i f_(i+1)^(i+1)+m_i p ̈_(c_i)^i (86)

 μ_i^i i = -f_i^i×(r_(i-1,i)^i+r_(i,C_i)^i)+R_(i+1)^i μ_i^i (i+1)^(i+1)+R_(i+1)^i f_(i+1)^(i+1)×r_(i,C_i)^i+(I_i^i $\int \omega_i^i + \omega_i^i = \sum_{i=1}^{n} \sum_{i=1}^{n} \omega_i^i$ + G_(sgr,i+1) (q ̊ _(i+1) I_(〖sg〗_(i+1)) z_(〖sg〗_(i+1))^i+q ̊ _(i+1) I_(〖sg〗_(i+1)) ω_i^i $\{sg\}$ $[(i+1))^i$) +3G_(pgr,i+1) (q $[i+1]$ I_($[\wp g]$ $[(i+1)]$ z $[[\wp g]$ $[(i+1)]^i$ +q $[(i+1)$ I_($[\wp g]$ $[(i+1)]$ $ω_i^i$ [']i×z₋($[pg]$ ₋(i+1))[']i) …………. (87)

The joint exerted at the joint (i) in current frame

τ_i^i=μ_i^(i^T) R_i^(〖i-1〗^T) z_0+G_(sgr,i) I_(〖sg〗_i) ω ̇_(〖sg〗_i)^(〖i-1〗^T) z_(〖sg〗_i)^(i-1)+3G_(pgr,i) I_($[pg]$] ω ($[pg]$] (i) ^($[i-1]$ ^T) z_($[pg]$ (i) ^ $(i-1)$ …………. (88)

In similar fashion, the equation of the joint torque of robot's link actuated by (n) equally planet-gears:

 $\tau_i^i = \mu_i^i(i^T) R_i^i$ π_i^j $\tau_i^j = 0 + G_{s_i^j} + G_{s_i^j} + G_{s_i^j}$ τ_i^j τ_i^j τ_i^j τ_i^j τ_i^j τ_i^j τ_i^j τ_i^j τ_i^j I_($[pg]$ i) ω ($[pg]$ i)^($[i-1]$ ^T) z_($[pg]$ i)^(i-1) …………. (89)

With μ_i^i =-f_i^i×(r_(i-1,i)^i+r_(i,C_i)^i)+R_(i+1)^i μ_i^i (i+1)+R_(i+1)^i f_(i+1)^(i+1)×r_(i,C_i)^i+(I_i^i $\int \omega_i^i + \omega_i^i = \sum_{i=1}^{n} \sum_{i=1}^{n} \omega_i^i$) $\omega_i^i = \sum_{i=1}^{n} (g_i^i + 1)$ $\omega_i^i = \sum_{i=1}^{n} (g_i^i + 1)$ $\sum_{i=1}^{n} (g_i^i + 1)$ $\sum_{i=1}^{n}$ ω_i^i i×z_(〖sg〗_(i+1))^i)+(n)G_(pgr,i+1) (q ̄_(i+1) I_(〖pg〗_(i+1)) z_(〖pg〗_(i+1))^i+q ̄_(i+1) I_(〖pg〗_(i+1)) ω_i^i×z_(〖pg〗_(i+1))^i) …………. (90)

3. Dynamic Simulation

As an example of simulation, we choose an elbow down of 2-R planar robot as shown Fig. 5. The links of robot have the same length (0.4m) and actuated by 3, 4, 5 and 6 equally spaced planet-gears as shown Fig. 6. The simulation of the kinetic energy and the joint torque can be done by using the mass properties shown in Table. 1. The links of robot, sungear, planet-gears and external mass are designed by using SolidWorks (2014) in such way the inertia tensor of link, sun-gear, planet-gears and external mass has a diagonal matrix about its center of mass in current frame, then the inertia tensor of the augmented link is also diagonal matrix about its overall center of mass $\left[\begin{array}{cc} \zeta \end{array}\right]$ in current frame.

The 2-R robot moves between two positions during 1s ($x=0.8$ m, $y=0$ m) to ($x=0.4499$ m, $y=0.5864$ m) with jerk zero at start-stop path and gives the trajectory shown in Fig (7) and it's verified by Matlab/simulink. Generation of trajectories with a bounded value of jerk improves the tracking accuracy and will allow to reach a higher speed of task execution, with eventually a reduction in the excitation of the resonant frequencies and the vibrations caused by the planetary gear system [30, 31, 32, 33]. We can compare between the two postures of 2-R robot by using the equations of the serial planar manipulators [34] in the dynamics modeling shown in section. 2 to choose the posture that has the power saving [35].

Figure 5 2-R robot actuated by planetary gear system modeled by using SolidWorks

3.1. Kinetic energy simulation

Figure 6 The kinetic energy of robot actuated by 3 ESPG

48.51 1.70 $rac{1}{2}$
 $rac{1}{2}$ 36.01 ම ē 1.27 $\begin{array}{c}\n\text{Use the equation: } \n\begin{aligned}\n\text{SVD2} & = 0.5 \\
\text{SVD2} & = 0.5 \\
\text{Total Line} & = 0.5$ Energy⁻ 0.81 Kinetic 0.40 $\overline{\mathsf{total}}$ 0.00 0.00 0.00 0.10 0.20 0.30 0.40 0.50 0.60 0.70 0.80 0.90 1.00

Figure 7 The kinetic energy of robot actuated by 3 ESPG by using SolidWorks

Figure 8 The kinetic energy of robot actuated by 4 ESPG

Figure 9 The kinetic energy of robot actuated by 4 ESPG by using SolidWorks

3.2. Joint torques simulation

Figure 12 The variation of joint torques of links of robot actuated by 4 ESPG

Figure 13 The variation of joint torques of links of robot actuated by 4 ESPG by using SolidWorks

4. Discussion

The dynamics modeling of robot actuated by (n) equally spaced planet-gears show that the kinetic energy of the augmented link shown in Eq. 57 and Eq. 63 (link+ sun-gear+planet-gears, link+external mass) is linear with respect to the dynamic parameters, namely, the mass, the three components of the first moment of inertia shown in Eq. 60 and Eq. 66, the six components of the inertia tensor shown in Eq. 62 and Eq. 68, and the moment of inertia of the sun-gear , planet-gears and external mass. The inertial effects of fast spinning sun-gear and planet-gears have a relevant influence on the dynamic behavior of manipulators. The effective sun-gear and planet-gears inertias are indeed multiplied by the square of gear ratios and the coupling torques and forces arise from the interaction with the link motion as shown in Eq. 57 and Eq. 89.

We can notice from the results obtained of the dynamic simulation of a 2-R robot shown in section. 3 that the results obtained, whether using simulation results or SolidWorks (2020) are the same, this similarity of results confirms the reliability and correctness of the dynamic model studied. [35-40]

Nomenclature

- \bullet C_i : Center of mass of augmented link.
- \bullet l_i : Center of mass of link (*i*).
- \bullet \bar{I}_i : Inertia tensor of augmented link (link+sun-gear+ planet-gears).
- \bullet \bar{I}_{i0} : Inertia tensor of augmented link (link+external mass).
- \bullet $I_{sg_{i+1}}$: Moment of inertia of sun-gear about its center of mass.
- \bullet $I_{pg_{i+1}}$: Moment of inertia of planet-gear about its center of mass.
- \bullet I_{e_m} : Moment of inertia of external mass about its center of mass.
- $\bullet \quad m_{e_m}$: Mass of external mass.
- m_{l_i} : Mass of link (*i*).
- \bullet I_{l_i} : Moment of inertia of link (*i*).
- \bullet $m_{sg_{i+1}}$: Mass of sun-gear.
- \bullet $m_{pg_{i+1}}$: Mass of planet-gears.
- \bullet m_i : Mass of augmented link (link+sun-gear+planet-gears).
- \bullet m_{io} : Mass of augmented link (link+external mass).
- \bullet r_{i-1,C_i} : Vector from origin of frame $(i-1)$ to center of mass C_i .
- r_{i, C_i} : Vector from origin of frame (*i*) to center of mass C_i .
- \bullet $r_{i-1,i}$: Vector from origin of frame $(i-1)$ to the frame (i) .
- \bullet r_{i-1,l_i} : The vector from origin of frame $(i-1)$ to center of mass of link (i) .
- r_{c_i, l_i} : The vector from the center of mass c_i to center of mass of link (*i*).
- $r_{C_i, sg_{i+1}}$: The vector from the center of mass C_i to the joint axis where the sun-gear is located.
- $r_{c_i,pg_{i+1}}$: The vector from the center of mass c_i to the center of mass of the planet-gear.
- r_{c_i, e_m} : The vector from the center of mass c_i to center of mass of external mass.
- \bullet C_{i} : Linear velocity of center of mass C_{i} .
- \ddot{P}_{C_i} : Linear acceleration of center of mass C_i .
- \cdot \dot{P}_i : Linear velocity of origin of frame (*i*).
- \bullet \ddot{P}_i : Linear acceleration of origin of frame (*i*).
- \bullet ω_i : Angular velocity of link (*i*).
- \bullet $\dot{\omega}_i$: Angular acceleration of link (*i*).
- \bullet ω_{sg_i} : Angular velocity of sun-gear.
- ω_{pg_i} : Angular velocity of planet-gear.
- \bullet ω_{sg_i} : Angular acceleration of sun-gear.
- \bullet $\dot{\omega}_{pg_i}$: Angular acceleration of planet-gear.
- \bullet g_0 : Gravity accelaration.
- f_i : Force exterted by link $(i-1)$ on link (i) .
- \bullet $-f_{i+1}$: Force exterted by link $(i + 1)$ on link (i) .
- μ_i : Moment exterted by link $(i-1)$ on link (i) with respect to origin of frame $(i-1)$.
- \bullet μ_{i+1} : Moment exerted by link $(i + 1)$ on link (i) with respect to origin of frame (i) .
- P_{C_i} : Vector from base coordinate system (x_0, y_0, z_0) to position of center of mass of augmented link (*i*).
- P_{l_i} : Vector from base coordinate system (x_0, y_0, z_0) to center of mass of link (*i*).
- \dot{p}_{l_i} : Linear velocity from base coordinate system (x_0, y_0, z_0) to center of mass of link (*i*).
- $+P_{sg_{i+1}}$: Vector from base coordinate system (x_0, y_0, z_0) to the joint axis where the sun-gear is located.
- \bullet $P_{pg_{i+1}}$: Vector from base coordinate system (x_0, y_0, z_0) to the center of mass of planet-gear.
- \bullet $\dot{p}_{sg_{i+1}}$: Linear velocity from base coordinate system (x_0, y_0, z_0) to the joint axis where the sun-gear is located.
- \bullet $\dot{p}_{pg_{i+1}}$: Linear velocity from base coordinate system (x_0, y_0, z_0) to the center of mass of planet-gear.
- P_{e_m} : Vector from base coordinate system (x_0, y_0, z_0) to center of mass of external mass.
- \bullet e_m : The center of mass of external mass.
- r : The radius of external mass.
- \bullet r_{e_m,c_i} : The vector from the center of mass of external mass to center of mass of augmented link (*i*).
- \bullet $S:$ Skew symetric matrix.
- R_i : The rotation matrix from Frame (*i*) to base frame (x_0, y_0, z_0).
- \bullet $G_{sgr_{i+1}}$: The gear reduction ratios between sun-gear and ring-gear.
- \bullet $G_{pgr_{l+1}}$: The gear reduction ratios between planet-gear and ring-gear.
- Z_{ra} : The number of ring gear teeth.
- Z_{sa} : The number of sun-gear teeth.
- $Z_{\nu q}$: The number of planet-gear teeth.
- T_{l_i} : The kinetic energy of link (*i*).
- \bullet $T_{sg_{i+1}}$: The kinetic energy of the sun-gear.
- \bullet $T_{pg_{i+1}}$: The kinetic energy of the planet-gear.
- \bullet T_i : The Kinetic energy of the augmented link.

Abbreviation

ESPG: Equally spaced planet-gears.

5. Conclusion

The dynamics modeling of robot holding an external mass and actuated by (n) equally spaced planet-gears are studied. The verification of the dynamic modeling of 2-R robot actuated by (n) equally spaced planet-gears by using the software SolidWorks (2020) permitted us to qualitatively develop and highlight the relevance of the dynamic model studied. These results are verified for 3-R planar robot and 3-R PUMA robot, and the dynamic modeling shown in section (2) are also verified by using Lagrange formulation.

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